

New subgrid-scale models for large-eddy simulation of Rayleigh-Bénard convection

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Rayleigh-Bénard convection (RBC)

- **Definition:** a convective cell heated from below and cooled from above.
- Flow dynamics are characterized by:

Rayleigh number:

$$Ra = \frac{g\alpha H^3 \Delta T}{\nu\kappa}$$

Prandtl number:

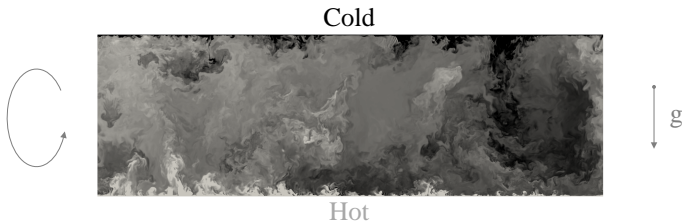
$$Pr = \frac{\nu}{\kappa}$$

cell aspect ratio Γ .

- system respond

$$Nu = \sqrt{RaPr} \langle wT \rangle - \frac{\partial \langle T \rangle}{\partial z}$$

- Clashes of Nu scaling in the ultimate regime $Ra > 10^{14}$.



Details of direct numerical simulation

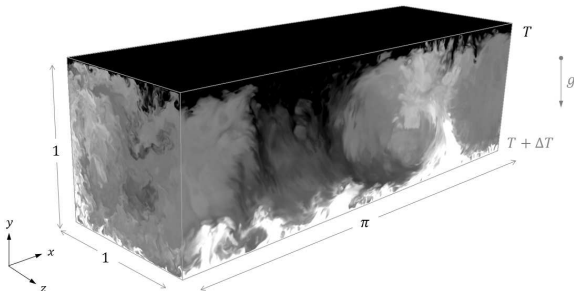
Dimensionless governing equations

$$\begin{aligned}\partial_t \mathbf{u} + (\mathbf{u} \cdot \nabla) \mathbf{u} &= -\nabla p + (Pr/Ra)^{1/2} \nabla^2 \mathbf{u} + \mathbf{f} \\ \partial_t T + (\mathbf{u} \cdot \nabla) T &= (RaPr)^{-1/2} \nabla^2 T\end{aligned}$$

Parameters

$Pr = 0.7$ (air flow),
 adiabatic sidewalls in z
 direction and periodic
 boundaries in x -direction.

- Refinements on Grötzbach criterion [Grötzbach, 1983] and coarsing approach in x -dir.
- 4th-order symmetry-preserving spatial discretizations [Verstappen & Veldman, J. Comp. Phys. 2003].



Case	Ra	$N_x \times N_y \times N_z$	N_{BL}	$t_{avg}[TU]$	Nu
DNS1	10^8	$400 \times 208 \times 208$	9	500	30.9
DNS2	10^{10}	$1024 \times 768 \times 768$	12	200	128.1

LES

- DNS is not feasible at very high Ra .

LES filtered equations

$$\begin{aligned}\partial_t \bar{\mathbf{u}} + (\bar{\mathbf{u}} \cdot \nabla) \bar{\mathbf{u}} &= -\nabla \bar{p} + (Pr/Ra)^{1/2} \nabla^2 \bar{\mathbf{u}} + \bar{\mathbf{f}} - \nabla \cdot \boldsymbol{\tau}(\bar{\mathbf{u}}) \\ \partial_t \bar{T} + (\bar{\mathbf{u}} \cdot \nabla) \bar{T} &= (RaPr)^{-1/2} \nabla^2 \bar{T} - \nabla \cdot \mathbf{q}(\bar{\mathbf{u}}, \bar{T})\end{aligned}$$

Where $\bar{(\cdot)}$ are the filtered resolved fields at filter length Δ .

$\boldsymbol{\tau}(\bar{\mathbf{u}})$ is the SGS stress tensor and $\mathbf{q}(\bar{\mathbf{u}}, \bar{T})$ the SGS heat flux vector.

Following the eddy-viscosity models and the gradient-diffusion hypothesis

- $\boldsymbol{\tau}(\bar{\mathbf{u}}) \approx -2\nu_t \mathbf{S}(\bar{\mathbf{u}})$
- $\mathbf{q}(\bar{\mathbf{u}}, \bar{T}) \approx -\kappa_t \nabla \bar{T}$

- $\mathbf{S}(\bar{\mathbf{u}}) = 1/2(\nabla \bar{\mathbf{u}} + \nabla \bar{\mathbf{u}}^t)$ is the rate-of-strain tensor.
- ν_t is the eddy viscosity.
- κ_t is the turbulent diffusivity.

Eddy-viscosity models and turbulent Prandtl number

ν_t is calculated following S3PQR models recently proposed by [Trias, PoF, 2015]:

- $\nu_t^{S3PQ} = (C_{s3pq}\Delta)^2 P_{GG^t}^{-5/2} Q_{GG^t}^{3/2}$
- $\nu_t^{S3PR} = (C_{s3pr}\Delta)^2 P_{GG^t}^{-1} R_{GG^t}^{1/2}$
- $\nu_t^{S3QR} = (C_{s3qr}\Delta)^2 Q_{GG^t}^{-1} R_{GG^t}^{5/6}$

- P_{GG^t} , Q_{GG^t} and R_{GG^t} are the first, second and third invariants of GG^t , where $G \equiv \nabla \bar{\mathbf{u}}$.
- C is the model constant.

κ_t is derived from ν_t by a constant turbulent Prandtl number $Pr_t = \nu_t / \kappa_t$

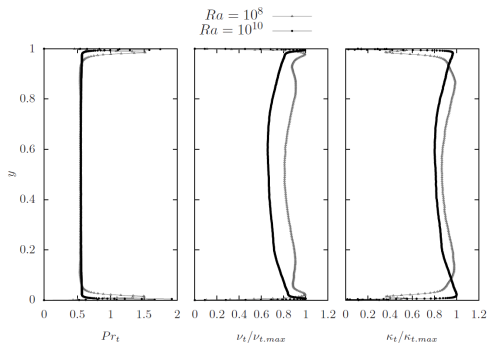
a priori Pr_t investigation

Using Taylor series expansion and the least square minimization technique:

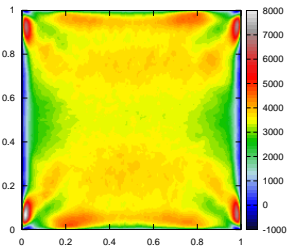
$$\nu_t \approx - \left\langle \frac{\mathbf{A}\mathbf{A}^t : \mathbf{S}(\mathbf{u})}{2\mathbf{S}(\mathbf{u}) : \mathbf{S}(\mathbf{u})} \right\rangle, \quad \mathbf{A} \equiv \nabla \mathbf{u}$$

$$\kappa_t \approx - \left\langle \frac{\mathbf{A}\nabla T \cdot \nabla T}{\nabla T \cdot \nabla T} \right\rangle$$

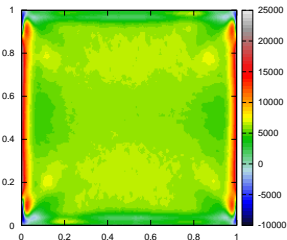
Turbulent Prandtl number



- Constant value of $Pr_t = 0.55$ in the bulk, independently of Ra .
- The turbulent wind is driven by the **mean buoyant forces** and not by the Reynolds stresses and ν_t is **positive**.



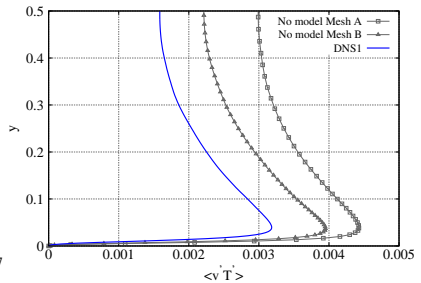
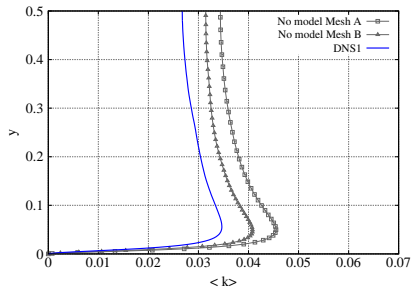
(a) Averaged t and x -dir ν_t



(b) Averaged t and x -dir κ_t

LES results at $Ra = 10^8$

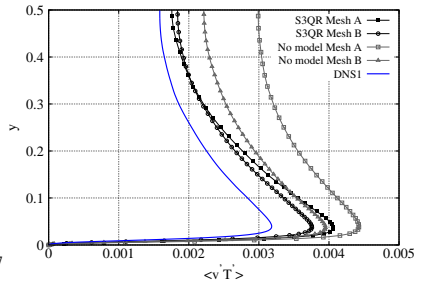
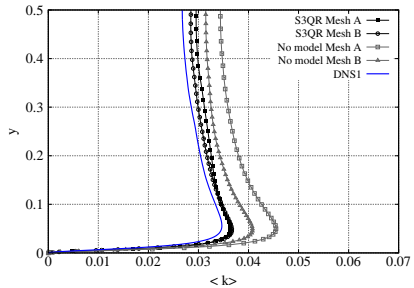
	DNS1	Mesh A	Mesh B
Mesh	$400 \times 208 \times 208$	$120 \times 80 \times 80$	$168 \times 110 \times 110$



- Turbulent kinetic statistics are good predicted in near-walls and bulk regions. However, an overestimation of heat flux within the boundary layer.
- One improvement option is the dynamical calculation of Pr_t .

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Use of velocity-times-temperature gradient tensor

Direct following to the evolution of **thermal plumes** and buoyant production using invariants of tensor $A_\theta = \nabla(\mathbf{u}T)^\dagger$:

- The square magnitude $|A_\theta|^2 = A_\theta : A_\theta = \text{tr}(A_\theta A_\theta^t)$

Hence

$$G_\theta G_\theta^t = \overline{T}^2 G G^t + 2\overline{T} \overline{\mathbf{u}} \otimes G \nabla \overline{T} + |\overline{\mathbf{u}}|^2 \nabla \overline{T} \otimes \nabla \overline{T}^t$$

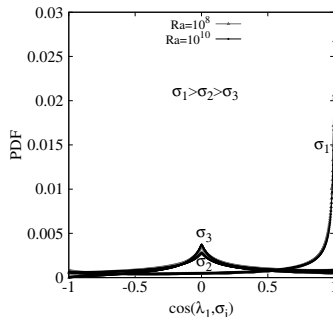
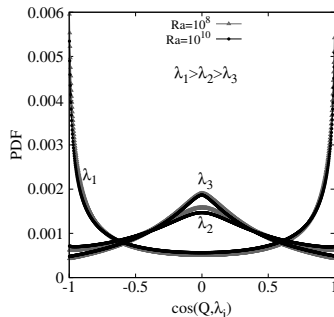
with $G_\theta = \nabla(\overline{\mathbf{u}}\overline{T})$, can be used in modeling the SGS heat flux.

- In the gradient model framework, it includes terms of Reynolds stress $G G^t$, turbulent heat flux $G \nabla \overline{T}$ and temperature variance $\nabla \overline{T} \otimes \nabla \overline{T}^t$.

[†]Dabbagh F., Trias F.X., Gorobets A. and Oliva A. "On the evolution of flow topology in turbulent Rayleigh-Bénard convection". Physics of Fluids (submitted)

Use of velocity-times-temperature gradient tensor

- Symmetric second-order tensor with **positive** eigenvalues and invariants.
- Has interactions of the small and large scales effects (**Buoyant and kinetic productions interchange**)



Invariants of $G_\theta G_\theta^t$ from instantaneous DNS fields at $Ra = 10^8$

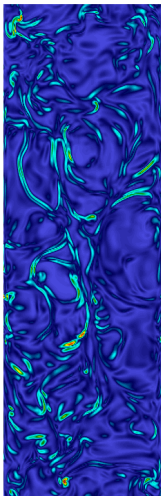


Figure: $G \nabla T$

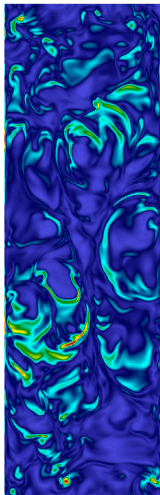


Figure: $P_{G_\theta G_\theta^t}$

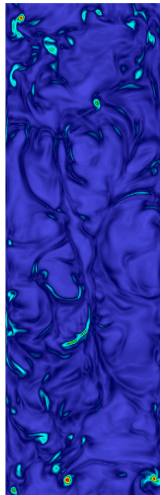


Figure: $Q_{G_\theta G_\theta^t}$

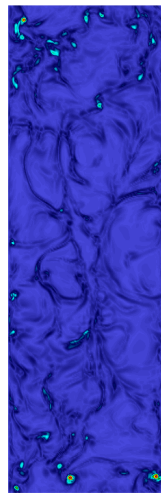


Figure: $R_{G_\theta G_\theta^t}$

conclusions

- DNS have been performed for turbulent RBC at $Ra = 10^8$ and 10^{10} .
- Turbulent Prandtl number has shown a **constant** bulk-dominated value $Pr_t = 0.55$ and the turbulent wind is driven by the mean buoyant forces at the sidewalls.
- The **S3QR** model proposed by [Trias, PoF, 2015], in LES frame modeling, has been applied in RBC.
- Good prediction of turbulent kinetics but overestimation of turbulent heat flux is overestimated near the wall.
- Encouraging use of $G_\theta G_\theta^t$ tensor in modeling the SGS heat flux.

Thanks for
your attention!